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DESIGN OF EXPERIMENT ANALYSIS OF AN ELECTRONICS PACKAGE  
LID USING FINITE ELEMENT ANALYSIS

BY

NUPUR BAJAD

BE, PUNE UNIVERSITY, 2012  
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THESIS

Submitted in partial fulfillment of the requirements for  
The degree of Master of Science in Industrial & Systems Engineering  
in the Graduate School of  
Binghamton University  
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The degree of Master of Science in Industrial & Systems Engineering  
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## Abstract

A design of experiment analysis is reported on data from warpage simulations using finite element analysis of a lidded electronics package. Warpage in a lid of an optical electronics package can detrimentally affect the reliability of the package as well as its optical performance. The present study focuses on the variety of materials and designs of lids relevant to recent technologies in electronics packaging. The finite element analysis (FEA) formulation in this study accurately predicts deformation and warpage in the elastic region with optimal computational time achieved through a choice of boundary conditions and mesh sensitivity studies.

The results from FEA are compared to analytical calculations made using the classical laminate plate theory (CLPT) as well as the modified Suhir's theory. It is observed that FEA results are more accurate as they account for the performance of die attach/underfill materials regardless of the small thickness of the layer. The FEA data are finally used to conduct a design of experiments (DOE) analysis to investigate the influence of 3 distinct designs and 6 material choices on warpage of a lid. The analysis indicates that there is no significant interaction between the two parameters expected to affect the warpage in the lid. Material properties of the lid are found to have a greater effect on the warpage of the lid as compared to variabilities introduced in lid designs in this study. The FEA simulations performed consider only material behavior within the elastic limit and, in some situations, plastic deformation may occur which is more permanent and as such requires a more comprehensive analysis in the plastic region to enhance the data set for DOE studies.

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# 1. Introduction

## 1.1. Overview

Warpage, which is mainly considered as out-of-plane deformation of an electronics package, is deformation that crucially affects the reliability of the surface mount packages. During a series of manufacturing processes of the electronic packages, such as printed wire board fabrication, solder paste printing, solder reflow process, etc., different thermomechanical stresses are generated in the package. Sometimes these stresses lead to fatigue and, eventually, cracks in the solder joints or other defects. These stresses can also lead to warpage of the package which, as noted above, can adversely affect reliability in terms of leading to solder joint defects as well as die cracking and delamination.

According to JEDEC standards, package warpage is the maximum distance between the contact plane and the bottom package surface within the measurement area [JESD22-B112A]. Warpage is mainly identified as one of two types, namely, convex or concave. The convention of the warpage is explained in Figures 1.1 and 1.2.

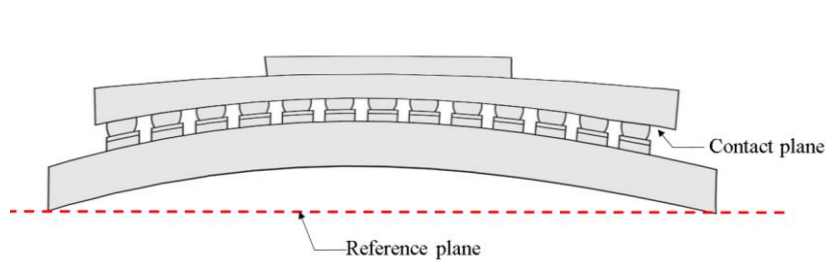


Figure 1.1: Convex (+) warpage

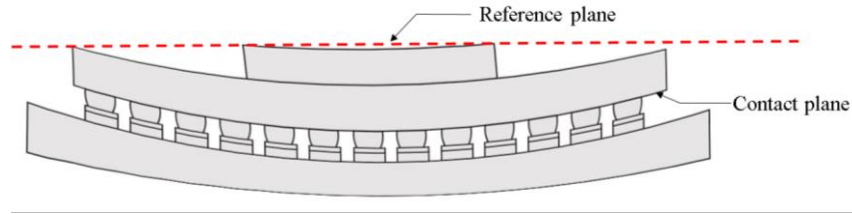


Figure 1.2: Concave (-) warpage

## 1.2. Techniques available to measure the warpage

There are different techniques available to measure the warpage of electronic packages such as experimental, analytical and simulation. Experimental techniques are known for more accurate measurements than analytical and simulation because of existing constraints of the each method. Experimental techniques of warpage measurement can be categorized into two types, namely, contact and non-contact. The gauge indicator shim method and contact profilometry method are of the contact type, whereas non-contact profilometry optical interferometry and digital image correlation are the non-contact types of techniques. Each technique has its own advantages and disadvantages. Gauge Indicator shim method is simple and inexpensive but manually operated so prone to errors induced by a human operator, and it is difficult to automate. This method also measures the rough warpage only along the edges and corners of the specimen with low resolution and low accuracy. Contact profilometry method has high resolution and accuracy since it is not sensitive to environmental noise, vibrations, surface reflections and contamination. Due to this, it is suitable for transparent and reflective surface measurements. The main disadvantage of this method is that it is highly time-consuming to get full field topology of the surface. Furthermore, this method occupies more space and is expensive because of the accessories to move stage by controlled motors and sensors. This mechanical movement

reduces the accuracy in terms of calibration. Non-contact profilometry is based on laser triangulation principle which has simple computation. Although this method can provide resolution up to sub micrometer scale it has few disadvantages such as it is unsuitable for inline measurements, mechanical movement reduces the calibration accuracy just as contact profilometry. While measuring the warpage on a rough surface, the resolution is limited due to laser speckle effect. Equipment based on optical interferometry has the advantage of high resolution, but they fail for samples which do not have a mirror like surface finish or big in size. Some equipment like electronic speckle pattern interferometry (ESPI) is sensitive to surface contamination and noise due to vibration.

Based on the resolution and application, an appropriate measurement technique can be selected. For this effort, the digital image correlation (DIC) technique, which is a non-contact method, is proposed. Figure 1.3 (a) shows schematic of the setup used in DIC method. DIC is based on the principle of stereo triangulation. Figure 1.3(b) shows the triangulation setup with respect to cameras. This technique needs two or more calibrated cameras which are constrained to have any relative motion. Calibration is done by having successive images of the target including the tilt, rotation, and movement of the target throughout the field of view. Precaution should be taken at this step to avoid deformation to the target. Triangulation angle between cameras maintained between  $10^{\circ}$ - $30^{\circ}$ . By using image matching techniques, the correspondence between object points is achieved. The surface is either etched or spray painted to obtain the pattern for reliable matching [JESD22-B112A]. This setup can achieve 1- $\mu\text{m}$  scale resolution with high measurement speed with no mechanical movement. This can be used for different sizes. There are certain disadvantages such as image distortion is possible due calibration inaccuracy. Results are

sensitive to surface reflectance and contamination. Sample preparation is complex. But due to its plain strain measurement capability and simple hardware with high measuring speed this method is widely used.

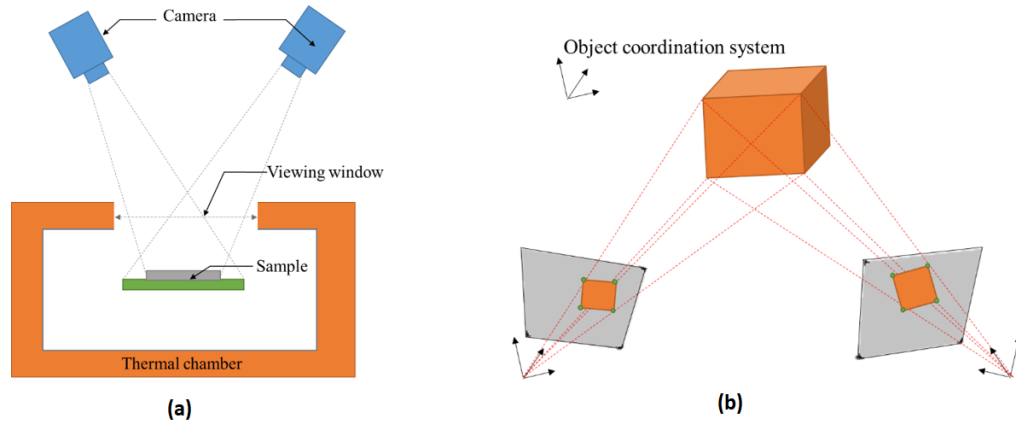


Figure 1.3: (a) Digital image correlation setup, (b) Triangulation setup

In addition to JEDEC having applied shadow moiré, digital image correlation and, fringe projection techniques to measure the warpage, it is utilized in this effort because of it has high resolution along with simple hardware and high speed data acquisition which is helpful for measurement of out-of-plane as well as in plane deformations of large areas [Kang & Ume, 2013].

Shadow moiré technique is known and most commonly used method for out-of-plane deformation measurement. It is first used by Ume et al. in 2002 for thermally induced warpage of PWB. Later, Many researchers have used this method for measuring warpage on BGA packages [C.Hartsough et al. 2007/ J.Pan et al. 2007/ B. Schwarz 2007], flexible circuit boards [L.Arruda et al. 2010], POP packages [Yang et al. 2005]. The fundamental principle of the shadow moiré technique on which most of the other techniques are based can be explained by using Figure 1.4. Generally, the tool that is used to measure the

warpage consists of a camera, light source, stage, a computer controlled user interface which can display fringe pattern, calibration, thermal chamber, ruled grating, thermocouple, and other components. The light source is fixed at an approximately 45° so that it is projecting a light beam through the grating. Under this grating, the surface of the specimen is exposed. Due to the geometric interference pattern created between the reference grating and the shadow grating, moiré fringe patterns are achieved. The resolution depends on the pitch of the grating.

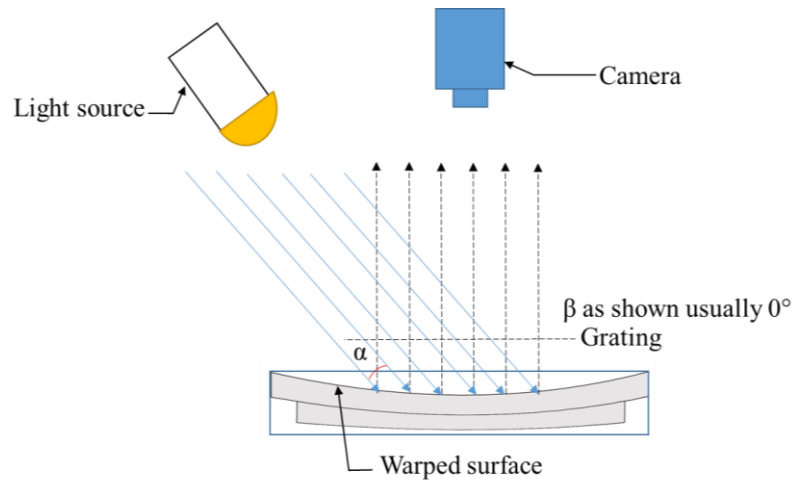


Figure 1.4: Experimental setup for the Shadow Moiré method

Out-of-plane deformation is calculated using Equation 1.1,

$$W = \frac{N \times p}{\tan\alpha + \tan\beta} \quad (1.1)$$

Where,

$W$  = out-of-plane (normal) displacement or warpage;  $N$  = Fringe order;  $p$  = grating pitch;

$\alpha$  = angle of illumination, and  $\beta$  = angle of observation.

Shadow moiré method has proven most suitable by Yeh et al. in the year 1991 since it is easy to use and less expensive, the ability for full field measurement, but during the analysis, the glass grating is located very near to the surface of the sample. This may affect the thermal behavior of the sample. Mechanical moving is another issue in this method where calibration accuracy is decreased. If the sample has shiny surfaces then usually it is painted in a white color to produce fringe.

Along with the experimental measuring tools, other methods can help to measure the deformation that is known as analytical techniques. For example, an analytical model that uses classical laminate plate theory (CLPT) is also utilized in the electronics packaging community. Herein, the package is considered as a composite with ‘N’ number of layers, comprised of different materials, and the technique considers the temperature dependent properties of the different materials. Simplified assumptions based on kinematic constraints are applied to multi-layered laminates to provide the thermo-mechanically produced displacement field at the mid-plane which essentially is considered as warpage [Park et al. 2007]. Furthermore, there is a large body of work that incorporates finite element analysis (FEA) to deduce warpage [Ke et al. 2009/ Jang et al. 2012 /Xiong et al. 2009]. In this method, which is simulation-based, a prototype is generated with a resemblance to the actual product and deformation is measured/estimated when subjected to a simulated thermomechanical load.

### **1.3. Research Objective**

The research objective of this work is to develop a DOE analytical method focusing on improving the performance of an electronic package lid. A sub-objective of this work is to try to understand the effect of material and design shape on warpage of the lid by



performing parametric analysis. This is important for technically advanced packages as shown in Figure 1.6 where purpose of the lid is not only protection and separation of the system boundary from surrounding but also provide cavity to hold the gas and optical path. In such cases, small amount of warpage in lid may cause failure to entire package.

For past research work solder joints are the considered to be most fragile parts of an electronics package and solder joint reliability is affected by package warpage, which is why a primary motivation for studies of the past to understand the reliability of the package by understanding the effect of warpage on solder joints.

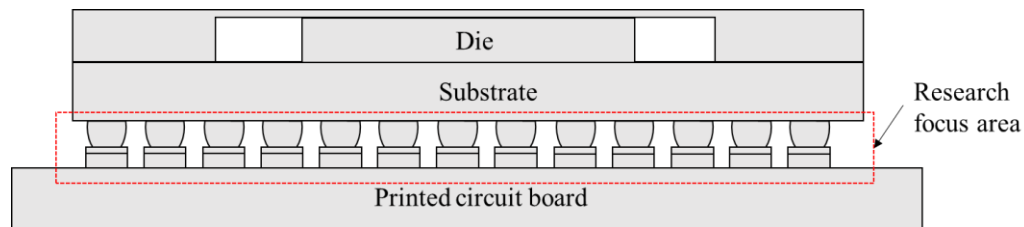


Figure 1.5: Schematic of focus of research area

There have been detailed studies conducted to understand the nature of the warpage, Yuan Li, 2003, C. Singh, 2010, Chang et al. 2008, etc. have studied warpage of different packages in recent years. In these studies, different effects of the warpage mainly focused on the solder joints of BGA packages or the interconnects of the different layers of the package. Figure 1.5 shows the area of focus in research for electronics packages using schematic. In 2014, Jang et al. have studied the warpage behavior and reliability of the chip-on flex package for thermal loading using FEA and Shadow moiré methods. All the studies in the past have used different techniques to measure the warpage since different package types may benefit from specific warpage measurement techniques.

Although experimental methods provide more realistic measurements of the deformation/warping, they may have constraints of expensive hardware, limitation over skills to operate, calibration accuracy, possibility to introduce human error. Similarly, finite element analysis method has the advantage of creating a prototype and solve under different loading conditions with less time but validation of the method is necessary and skills to operate software along with the cost to use this commercially available software. An analytical method has limitations for providing a suitable solution for all types and designs of the package. The analytical method also focuses more on estimating warpage on the mid plane of the package where as in simulation focus of the study can be associated with different components of the package.

There is a wide variety of materials utilized in electronics packagings such as metals, plastics, liquid crystal polymers (LCP), and others. Depending on the application, packaging materials need to be wisely selected. For example, the shrinkage phenomenon is a concern for plastic materials and, thus, warpage can be a major concern. Shrinkage occurs due to the polymer varying its density when there is a change in the temperature. During injection molding, variation in the shrinkage creates residual stresses. These residual stresses play an important role in the warpage of the part since warpage occurs where the product does not follow the intended design. So, ideally, if the shrinkage is uniform throughout the part then the part may not warp. But, due to the interaction of many factors such as molecular orientations, mold cooling temperature, the design of parts, etc., it is difficult to achieve uniform shrinkage [Fischer J. 2013].

Currently, different designs are available for various sensors. Figure 1.6 exhibits a few commercially available lidded gas sensors by different vendors. A quick review of this

figure suggests that different designs of lids are considered, and the purpose of the lid is not limited to providing protection during or post-assembly.



Figure 1.6: Commercially available CO2 Sensor which uses lid as gas cavity

Two sensors shown on the left and right are different CO2 sensor modules made by SOHA Tech Co.,Ltd [Retrieved from: <http://sensor-co2.com/products/co2-module/?ckattempt=1>]

#### 1.4. Significance of research

This research mainly focuses on how warpage is affecting the lid deformation and techniques to characterize it. As discussed earlier, FEA tool is used in this study to create a prototype which is similar to the actual product. The experiment is designed considering different variables such as the design of the lid and the material of the lid. Methods of the design of experiment analysis are applied to understand the correlation between these parameters. The most significant parameter in terms of the warpage deformation is addressed. Based on this study the appropriate design and material are suggested for the development of the lid over the package. This becomes helpful when there is an optoelectronic package undergoing thermomechanical loading; warpage may not only

adversely affect solder joints but other parts of the package as well. So in this research work, characterization of the lid of the package affected by warpage is the focus area. This study will be helpful for the development of the technologically advanced packages associated with optoelectronics.

## **2. Literature Review**

As briefly discussed in Chapter 1, there are different methods to estimate the out-of-plane deformation when a package is subjected to thermomechanical loading conditions. These methods can be analytical, numerical or experimental. With respect to analytical techniques, Suhir's theory [1980] for analytical warpage prediction for die – substrate assembly and classical laminate plate theory (CLPT) by S.B. Park [2007] are well known. Finite element analysis (FEA) is an automated technique that can be used to estimate the deformation based on numerical calculations.

More details on the development of experimental methods are presented in Section 2.1 and those of some analytical methods are discussed in Section 2.2.

### **2.1. Experimental Methods**

As mentioned in Chapter 1 experimental methods are classified into contact as well as non-contact type. The following are some of the experimental methods utilized in the electronics industry to estimate warpage.

### 2.1.1. Gauge-Indicator Shim method

One of the oldest known measurement techniques of warpage measurement utilized in the electronics industry is a gauge–indicator shim method. This is a mechanical method where a filler gauge is placed under the PWB and has been discussed by Farrago in 1994 [Kang & Ume, 2013]. The filler gauge is similar to a spark-plug gap set of shims, but for electronics, the shims are much smaller. An example filler gauge is depicted in Figure 2.1.



Figure 2.1: Filler gauge set [“Deluxe Feeler Gauge.” Princess Auto, (n.d.). Retrieved June 02, 2017, from <http://www.princessauto.com/en/detail/deluxe-feeler-gauge/A-p2990117>]

Although this is a simple and inexpensive method, it currently does not see significant use because it measures warpage only along the edge and corner of the sample, with low resolution and low accuracy.

### 2.1.2. Contact profilometry

In this method, variation in surface profile is measured as a function of position of stylus. The stylus moves along the specified surface. A transducer is connected to the stylus which generates a signal as per the displacement in the vertical direction. This signal is later processed and converted to a surface profile. The radius of the diamond stylus varies

from 20nm to 25 $\mu$ m, which helps to achieve a high resolution. However, there is a practical limitation to achieving this high resolution. The method is time consuming to generate a full field topology of the surface. So, as compared to other methods, the practical resolution of warpage measurement is low. This method is not suitable for soft surfaces; other advantages of this method are accuracy, robustness and surface independence. So, measurement of warpage is not affected by the contamination on surface, reflectance properties of surface, nor with environmental noise, such as vibration.

The contact profilometry method has been used to measure warpage in various types of packages. Miyake [2001] have used this to measure warpage of thin small outline package (TSOP). Yang et al. [2005] have used this to measure process-induced warpage in quad flat nonlead (QFN) packages.

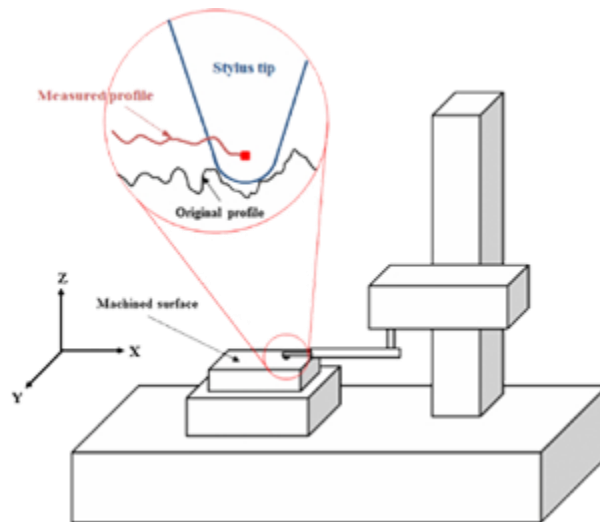


Figure 2.2: Schematic of stylus profilometer

[(n.d.). Retrieved October 20, 2017, from <http://www.nanoscience.com/technology/optical-profiler-technology/how-profilometer-works/>]

Experimental methods mentioned above are of contact type. In-line measurement capability for contact type of methods is zero. Some of these non-contact type methods are as discussed, below.

### **2.1.3. Non-contact profilometry**

This method is also called as optical profilometry because it uses an optical approach. The reflected light from the specimen is used to generate a 3D surface profile applying the triangulation principle as shown in Figure 1.3 (b) and discussed in Chapter 1 along with the digital image correlation (DIC) method.

### **2.1.4. Optical interferometry**

This method mainly uses the interference of the light waves for surface diagnostics. A wide variety of configurations is available in this type for warpage measurement of electronic packages. A few well-known configurations are Twyman–Green interferometry, Fizeau interferometry, and Electronic speckle–pattern interferometry. The basic principle of these methods is the same but there are differences in the optical setup to achieve the interference patterns. Application of Twyman–Green interferometry is limited since it requires a mirror-like surface. Also, the measurement sensitivity is within a fraction of a micron, which is sufficient to resolve the warpage observed in advanced electronic packages.

The setup of far infrared Fizeau Interferometry developed by B. Han [2003] to measure warpage of flip-chip packages is as shown in Figure 2.3. The laser beam is attenuated by a partial reflector. This transmitted light is expanded and collimated by a lens. Partial collimated light is reflected from an optical flat where the transmitted beam is



reflected from the specimen. Both of these reflected beams are collected by an infrared camera. This method is mostly used to measure samples of small size and the setup is complex and expensive.

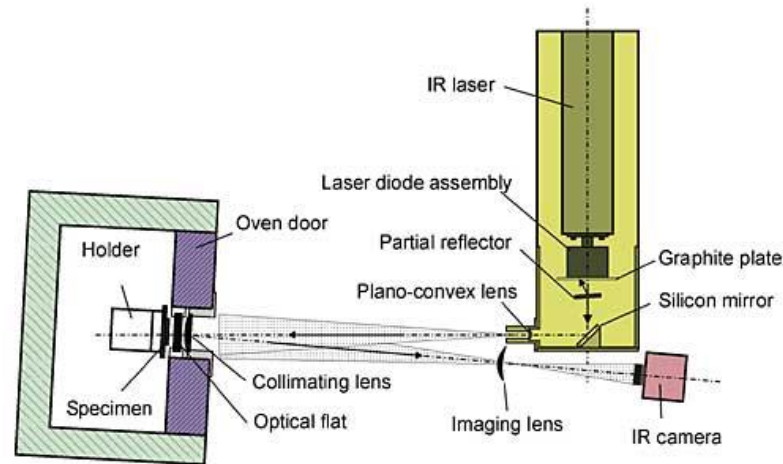


Figure 2.3: Optical configuration of the Fizeau Interferometry by Han [2003]

### 2.1.5. Moiré Methods

Moiré methods are widely known/utilized and are based on the fringe projections that can define the surface topology. The methods were introduced in 1967 by Rowe and Welford. Moiré fringes are generated by interference of two periodic grating lines, one is a reference, where the other is the specimen's grating lines. As the specimen deforms under load, the specimen grating lines also get distorted. This helps to get surface height variation experienced by the sample. Based on the location of the specimen grating, this method is characterized further in two types, namely, in-plane and out-of-plane moiré. In the case of in-plane moiré, the specimen grating is either attached or printed to the specimen surface and the reference grating is fixed with constant period and spatial orientation. In the case of out-of-plane moiré, which is also known as moiré topology, a fringe pattern is developed

by superimposing both the gratings. Specimen grating is adjusted and distorted with reference of out-of-plane displacement (warpage) of the specimen. The out-of-plane moiré method is more useful to measure warpage of the chip packages and PWB assemblies due to high resolution and capability to measure large areas.

Based on the different methods to generate and study fringe patterns the moiré method is classified as Shadow moiré, Projection moiré and Digital fringe projection. This is because the methods for generation of fringe evolved since successful use of shadow moiré method by Ume et al. at the Georgia Institute of Technology [2002] for measuring warpage of printed wire boards (PWB).

Kang S. & Ume I.C. [2013] have compared all these techniques of measuring warpage by scaling and putting a tangible value on the same scale for properties like resolution, cost, accuracy, capacity for in-line measurement, flexibility, and speed for measurement. Table 2.1 shows that all the properties are averaged on a 4.0 scale and outcome show that shadow moiré is a better method as compared to other methods.

Table 2.1: Comparison of the different techniques available for measuring warpage based on measurement features, Kang & Ume. [2013]

Features (4.0 Scale)	Contact Type		Non-contact Type							
	GIS	CP	NCP	Optical Interferometry			DIC	Moiré Methods		
				TGI	FI	ESPI		SM	PM	DFP
Resolution	0	2	3	4	2	3	3	4	2	3
Speed	0.7	1.3	1.3	2.3	2.3	2	3	3.3	3	2.3
Costs	4	1.7	1.3	2.7	2.7	2	2.7	3	2.7	2.7
Accuracy	0.5	2	2	3	2.5	2	2	3	2	2.5
Flexibility	3.5	3	2	1	2	1.5	2	4	2.5	3
Robustness	2	2.3	2.3	2	2.7	1.7	1.7	2.3	2.3	2.3

In-line capability	0	0	1.6	2	2.4	1.6	2.2	2.8	2.4	2.2
Average	1.5	1.8	1.9	2.4	2.4	2	2.4	3.1	2.4	2.6

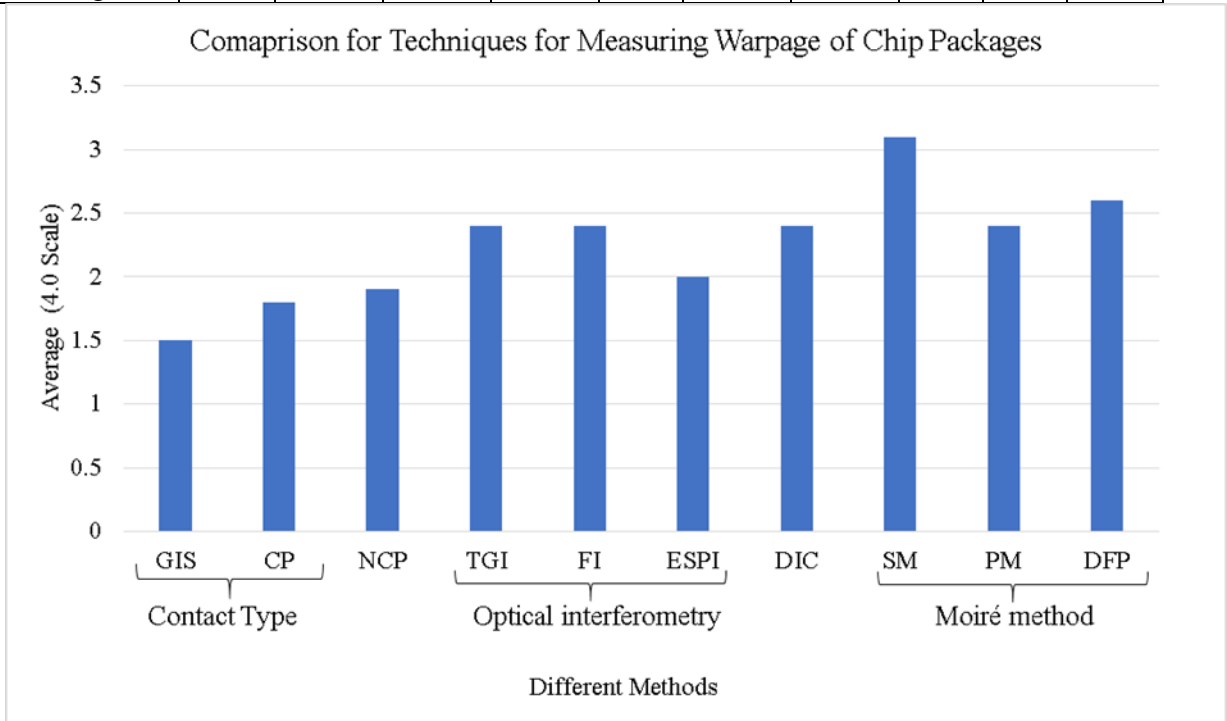


Figure 2.4: Graphical representation of the comparison Table 2.1

[GIS= Gauge indicator Shim method, CP= contact profilometry, NCP= Noncontact profilometry, TGI= Twyman –Green interferometry, FI= Fizeau interferometry, ESPI= Electronic speckle –pattern interferometry, DIC= Digital image correlation, SM= Shadow moiré, PM= Projection moiré, DFP= Digital fringe projection]

## 2.2. Analytical and numerical methods

### 2.2.1. FEA Analysis- Numerical method

The finite element analysis method was developed by R. Courant in 1943. The intent was to develop a method to solve vibration systems using the Ritz method of numerical mathematics and variational calculus. Due to time constraints of hand calculations for large-scale matrix transformation applications of FEA, the technique was

limited at that time. In early 1970, the FEA method was mainly used for aeronautics as well as automotive industries as based on the research done by Argyris et al. [1950] in design of airframe and some basic structural analysis of automobile chassis. Yet due to the lack of high computation performance and high price of mainframe computers, FEA was only applied to simple structural analysis. As computing became cheaper, coupled with increased computational speed, FEA gained tremendous use. Current software packages, including ANSYS, Flotherm, and others, offer comprehensive multi-physics solutions while considering different analytical conditions such as structural, thermal, fluid, and electromagnetic. In the area of electronics packaging, thermo-mechanical loading conditions play an essential role to define the reliability of the packages.

In the literature, many researchers have used the FEA method for solving thermo-mechanical analysis for different applications such as the study of crack propagation in solder joints, thermal performance of power electronics, and warpage behavior of the package considering different parameters. Figure 2.5 shows the wide variety of applications of the FEA tool in the semiconductor packaging industry. Figure 2.6 shows the flowchart for solving complex problems in FEA.

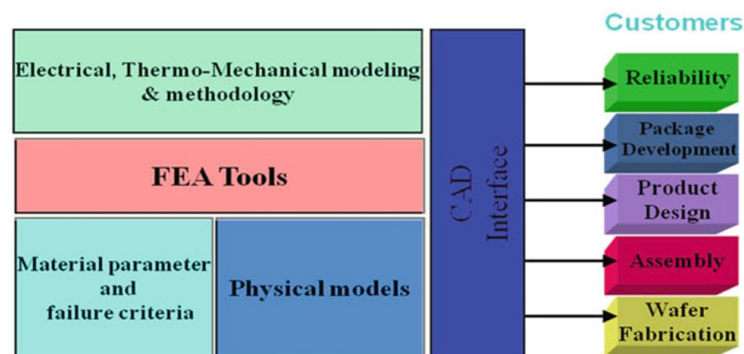


Figure 2.5: Basic mapping of modeling and simulation in semiconductor industry, image from Liu, Y. [2012]

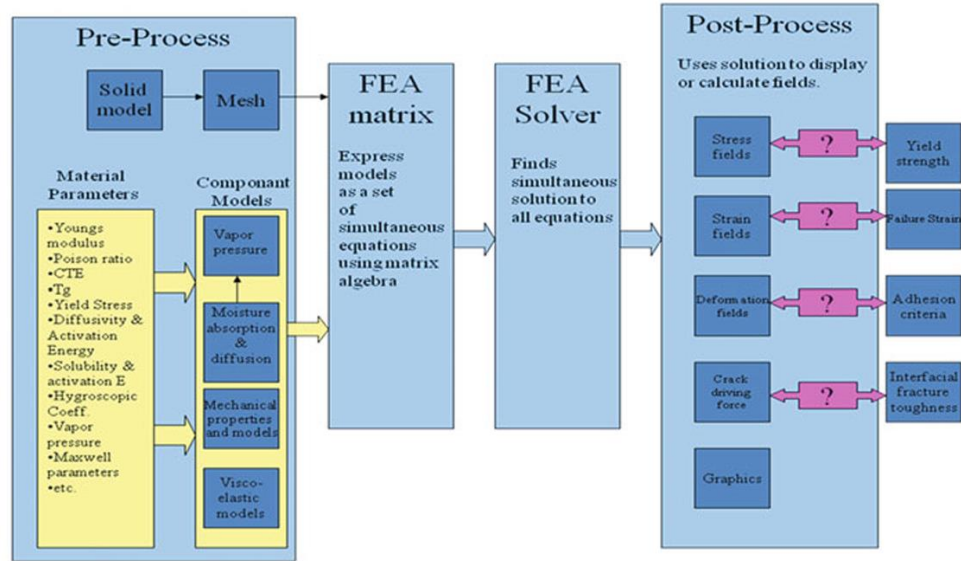


Figure 2.6: Flowchart for FEA method processing, image from Liu, Y. [2012]

In terms of academic research, FEA tools are used prominently to validate with the analytical method and experimental results. Sun et al. [2008] have conducted warpage simulations and design of experiment (DOE) analysis for package-on-package (POP) development. In their study, a land grid array (LGA) package is simulated under thermal conditions of ramp up from 175°C to 260°C and ramp down from 175°C to 25°C. In their simulation, Epoxy mold compound (EMC) was assumed as linear elastic even though it is viscoelastic in nature; the assumption was made since it was difficult for them to obtain the material properties from the supplier.

In this case study, the average CTE is calculated where glass transition temperature ‘ $T_g$ ’ is falling in the simulation temperature cycle range.

$$CTE_{ave} = \frac{\alpha_1(T_g - T_{final}) + \alpha_2(T_{ref} - T_g)}{T_{ref} - T_{final}} \quad (2.1)$$

The present work demonstrates an FEA analysis based on material properties, dimensions and other geometrical parameters from Sun et al. [2008]. This exercise helps to estimate correct boundary and symmetry conditions of the quarter model of entire package in order to save computational time for simulations in current studies.

### 2.2.2. Analytical method- Classical laminate plate theory (CLPT)

Classical laminate plate theory (CLPT) is a popular method that considers an “n” layer laminate of the sample where mechanics of the stresses and deformation is simplified in terms of a matrix that considers the elastic modulus, Poisson’s ratio, thermal expansion coefficient, and thickness of each layer.

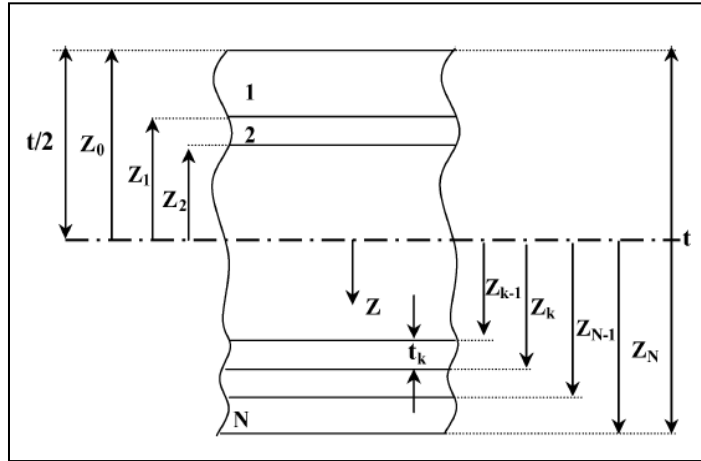


Figure 2.7: Geometry of laminate considering ‘N’ layers, image from Park et al. [2007]

Laminate warpage is deduced at the mid-plane by its extensional strains  $\{\varepsilon^0\}$  and its curvatures  $\{k\}$ . When the  $n^{\text{th}}$  layer laminate is subjected to temperature change, the stress strain relationship is established as –

$$\{\sigma\}_k = [Q]_k \{ \{\varepsilon\}_k - \{\alpha_k\} \Delta T \} , \quad (k = 1, 2, 3, \dots, N) \quad (2.2)$$

$$\text{Where, } [Q]_k = \frac{E_k}{1-\nu_k^2} \begin{bmatrix} 1 & \nu_k & 0 \\ \nu_k & 1 & 0 \\ 0 & 0 & \frac{1-\nu_k}{2} \end{bmatrix} \quad (2.3)$$

For calculating  $[Q]_k$ , the material is considered to be isotropic and  $E_k$  and  $\nu_k$  represent the elastic modulus and Poisson's ratio of the  $k^{\text{th}}$  layer, respectively. The relationship between plane stress  $\{\varepsilon^0\}$ , curvature  $\{k\}$ , and resultant forces  $\{N\}_\Lambda$  and moments  $\{M\}_\Lambda$  is given as –

$$\begin{Bmatrix} \varepsilon^0 \\ k \end{Bmatrix} = \begin{bmatrix} A & B \\ B & D \end{bmatrix}^{-1} \begin{Bmatrix} N_\Lambda \\ M_\Lambda \end{Bmatrix} \quad (2.4)$$

Where,

$$\{N_\Lambda\} = \sum_{k=1}^n \int_{z_{k-1}}^{z_k} [Q_{ij}]_k \{\Lambda_k\} dz = \sum_{k=1}^n [Q_{ij}]_k \{\Lambda_k\} (z_k - z_{k-1}) \quad (2.5)$$

$$\{M_\Lambda\} = \sum_{k=1}^n \int_{z_{k-1}}^{z_k} [Q_{ij}]_k \{\Lambda_k\} z dz = \frac{1}{2} \sum_{k=1}^n [Q_{ij}]_k \{\Lambda_k\} (z_k^2 - z_{k-1}^2) \quad (2.6)$$

$$\{\Lambda_k\} = \alpha_k \Delta T \begin{Bmatrix} 1 \\ 1 \\ 0 \end{Bmatrix} \quad (2.7)$$

$[A]$ ,  $[B]$ , and  $[D]$  stiffness matrices are,

$$[A] = \sum_{k=1}^n \int_{z_{k-1}}^{z_k} [Q_{ij}] dz = \sum_{k=1}^n [Q_{ij}]_k (z_k - z_{k-1}) \quad (2.8)$$

$$[B] = \sum_{k=1}^n \int_{z_{k-1}}^{z_k} [Q_{ij}] z dz = \frac{1}{2} \sum_{k=1}^n [Q_{ij}]_k (z_k^2 - z_{k-1}^2) \quad (2.9)$$

$$[D] = \sum_{k=1}^n \int_{z_{k-1}}^{z_k} [Q_{ij}] z^2 dz = \frac{1}{3} \sum_{k=1}^n [Q_{ij}]_k (z_k^3 - z_{k-1}^3) \quad (2.10)$$

Middle surface curvature ( $w_0$ ) can be calculated as follows-

$$\begin{bmatrix} k_x \\ k_y \\ k_{ky} \end{bmatrix} = - \begin{bmatrix} \frac{\partial^2 w_0}{\partial x^2} \\ \frac{\partial^2 w_0}{\partial y^2} \\ 2 \frac{\partial^2 w_0}{\partial x \partial y} \end{bmatrix} \quad (2.11)$$

And out-of-plane deformation is calculated as,

$$\omega = -\frac{1}{2} (k_x x^2 + k_y y^2 + 2k_{xy} xy) \quad (2.12)$$

Park et al. [2007] have used this method for prediction of warpage in the chip scale packages (CSP) and for a flip chip package with large substrate considered as package with overhang.

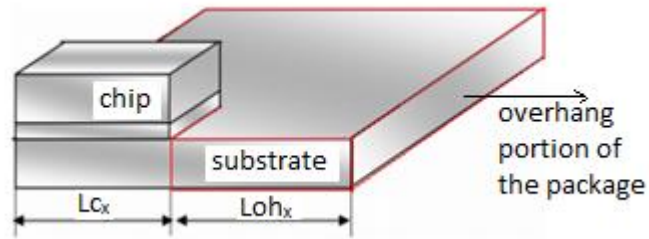


Figure 2.8: Geometry of Package highlighting the overhang, image from Park et al. [2007]



Out-of-plane deformation at the end of the chip in the X-direction is given by,

$$w_{c_x} = -\frac{1}{2}(k_x x^2) \quad (2.13)$$

For overhang portion, out-of-plane deformation in the X-direction is,

$$w_{oh_x} = L_{c_x} L_{oh_x} k_x \quad (2.14)$$

Total out-of-plane deformation in the X-direction is,

$$w_x = w_{c_x} + w_{oh_x} \quad (2.15)$$

Similarly, in the Y-direction,

$$w_y = w_{c_y} + w_{oh_y} \quad (2.16)$$

By using the formulae given above and FEA method, Park et al. [2007] have developed a predictive model for optimized design parameters in flip chip package and assemblies. In that work, they have studied the importance of applying appropriate effective moduli of the material when there is composite structure of solder joints covered in underfill.

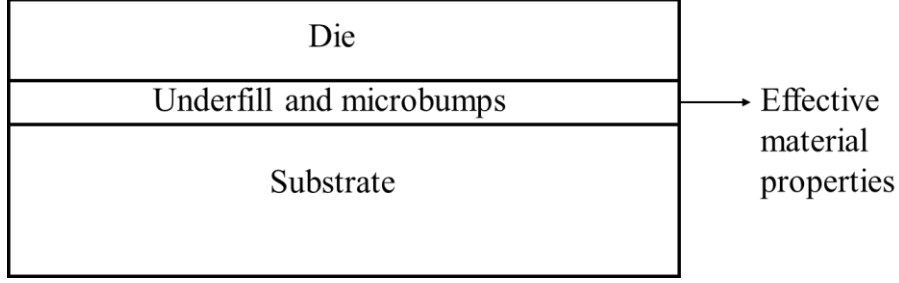


Figure 2.9: Geometry of Package highlighting the layer where effective material properties are to be considered

The effective material property of the composite layer is calculated using a simplified formula as follows,

$$E_{\text{effective}} = \frac{E_{\text{Solder}}V_{\text{Solder}} + E_{\text{Underfill}}V_{\text{Underfill}}}{V_{\text{Solder}} + V_{\text{Underfill}}} \quad (2.17)$$

$$\mu_{\text{effective}} = \frac{\mu_{\text{Solder}}V_{\text{Solder}} + \mu_{\text{Underfill}}V_{\text{Underfill}}}{V_{\text{Solder}} + V_{\text{Underfill}}} \quad (2.18)$$

$$\alpha_{\text{effective}} = \frac{\alpha_{\text{Solder}}E_{\text{Solder}}V_{\text{Solder}} + \alpha_{\text{Underfill}}E_{\text{Underfill}}V_{\text{Underfill}}}{E_{\text{Solder}}V_{\text{Solder}} + E_{\text{Underfill}}V_{\text{Underfill}}} \quad (2.19)$$

Where,  $E_{\text{effective}}$ ,  $\mu_{\text{effective}}$ ,  $\alpha_{\text{effective}}$  are elastic modulus, Poisson's ratio, and coefficient of thermal expansion (CTE) of the composite layer, respectively. And conclusion the study as for composite layer considered to be isotropic then in-plane effective moduli are more appropriate than out-of-plane.

### 2.2.3. Modified Suhir's solution

Later, a study presented by Tsai et al. [2009] suggested that classical lamination theory assumes that radius of curvature is constant with respect to the Kirchhoff- Love hypothesis. So, the formula mentioned above becomes invalid and provides more errors as

the die attach layer is more compliant. Tsai et al. [2009] suggested the modified Suhir's [1980] solution for simplified flip chip package represented in Figure 2.10.

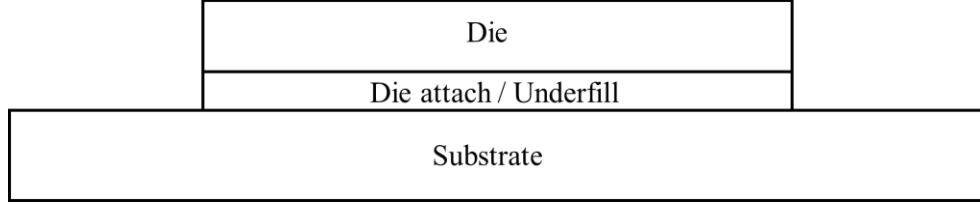


Figure 2.10: Schematic of simplified flip chip package

There are certain assumptions included in this theory; one such assumption, for example, is that isothermal loading is applied on the package and that there is a perfectly bonded interface between each layer. Also, each layer acts as a spherically bending thin plate. Due to there being a small thickness, along with the underfill layer and material being compliant with the die and substrate, the CTE of the underfill is insensitive to stresses and therefore neglected.

Warpage is ,

$$w(x) = \frac{t\Delta\alpha\Delta T}{2\lambda D} \left[ \frac{1}{2}x^2 - \frac{\cosh kx - 1}{k^2 \cosh kl} \right] \quad (2.20)$$

And radius of curvature is,

$$\frac{1}{\rho(x)} = \frac{t\Delta\alpha\Delta T}{2\lambda D} \left( 1 - \frac{\cosh kx}{\cosh kl} \right) \quad (2.21)$$

Where,  $\Delta T$  is difference between final temperature and initial temperature and  $\Delta\alpha$  is CTE mismatch between the adjacent layers of the package.

$$k = \sqrt{\frac{\lambda}{\kappa}} \quad (2.22)$$

$$\lambda = \frac{1 - \nu_{die}}{E_{die}t_{die}} + \frac{1 - \nu_{substrate}}{E_{substrate}t_{substrate}} + \frac{t^2}{4D} \quad (2.23)$$

Here,  $t$  is the total thickness and is flexural rigidity of the package, calculated as,

$$t = t_{die} + t_{die\ attach} + t_{substrate} \quad (2.24)$$

$$D = D_{die} + D_{die\ attach} + D_{substrate} \quad (2.25)$$

$$D = \sum \frac{E_i t_i^3}{12(1 - \nu_i)} \quad (2.26)$$

$$\kappa = \frac{t_{die}}{3G_{die}} + \frac{2t_{dieattach}}{3G_{dieattach}} + \frac{t_{substrate}}{3G_{substrate}} \quad (2.27)$$

Here,  $E$ ,  $\nu$  and  $G$  are, respectively, elastic modulus, Poisson's ratio, and shear modulus of the material. Based on their study it is concluded by Tsai et al. [2009] that low elastic moduli of the underfill can significantly reduce the maximum warpage and it was suggested to use either a low modulus of the underfill or low CTE for the substrate to reduce maximum warpage effect.

### 2.3. Summary

The literature available on the warpage measurements of the electronic packages mainly focuses on the warpage on the entire package, typically at the mid plane of the package. Taking into consideration the total thickness of the package, this mid plane lies closer to the PCB/substrate rather than the lid. There are limited studies that analyze or report measurements of warpage in the lid of an electronics package. In this research work,

performance of the package under thermo-mechanical loading in terms of warpage on the lid of the package is analyzed using a combination of FEA and DOE analyses. The effect of different parameters in the design process of the lid of the package are investigated using FEA simulations. These parameters include the material properties, shape and different thickness of the lid on the package. Dasgupta et al. [1998] have discussed an approach for design of experiment methods for parametric study in electronics packaging. In a subsequent chapter, a similar approach using design of experiment methods is applied to investigate the effect of different parameters on the lid.

### **3. Research Methodology**

The previous chapter discussed different methods of estimating warpage of electronics packages. Considering the availability of resources for this project, a combination of FEA analysis and analytical methods is employed. In formulating the FEA simulation a case study derived by Sun et al. [2008] has proven helpful to understand the correct boundary conditions and thermo-mechanical loading setup. In that study, the warpage of the package is measured across the diagonal to estimate accurately the magnitude as well as nature of the warpage. The details of the case study and further research work as follows.

#### **3.1. Research Experiment – Case study**

In order to collect data for design of experiments analysis, simulations were performed by using FEA software ANSYS 17.0. The research methodology is developed based on the case study solved as given by W. Sun et al. [2008]. An LGA package is simulated under thermal loading conditions of cooling from 175°C to 25°C and heating from 175°C to 260°C with stress-free state assumed at 175°C. Material properties and dimensions were considered as shown in Table 3.1.

Table 3.1: Dimensional details and Material properties of different layers in LGA package, Sun et al. [2008]

LGA layers	Length×Width	Thickness	Young's modulus at 25°C	Young's modulus at 260°C	CTE at 25°C	CTE at 260°C	T <sub>g</sub>
Unit	(mm)		(MPa)		(ppm/°C)		(°C)
Die	5×9.7	0.1	131000		2.6		-
Die attach	5×9.7	0.04	230	5	41	172	41
EMC	8×10.5	0.35	24000	280	9	36	135
SM		0.03	2400	800	60	130	100
Core		0.1	28000	8600	*XY:14,Z:30	XY:7, Z:150	220
Cu		0.02	11700		17.3		-

In ANSYS, a three-dimensional model is created in the X-Z plane and has thickness in the Y direction. CTE values for the orthotropic material as mentioned in Table 3.1 are assigned in terms of the Y direction while simulating the experiment. The geometry of the model created is shown in Figure 3.1-

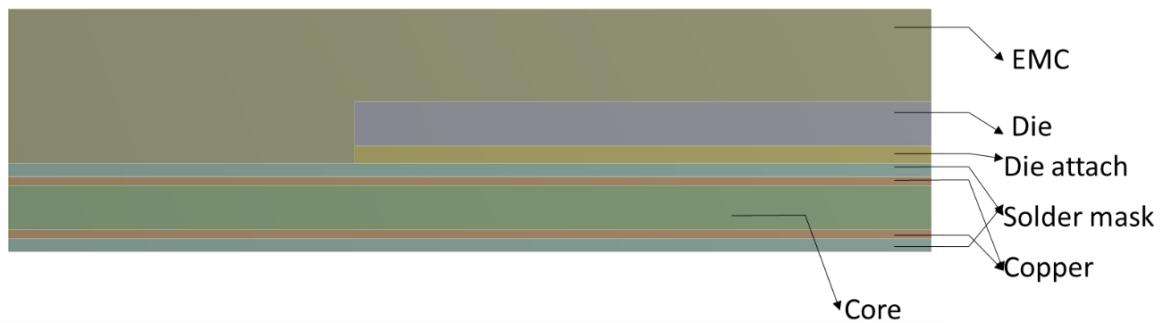


Figure 3.1: Front view of the cross section of quarter model of LGA showing layers categorized in different colors

A quarter meshed model is generated, implying that the geometry analyzed is only a quarter fraction of the entire model. This is done by exploiting the model's symmetry along the faces in the Y-Z and X-Y planes as shown in the Figure 3.2. This significantly reduces computational time to solution of the model.

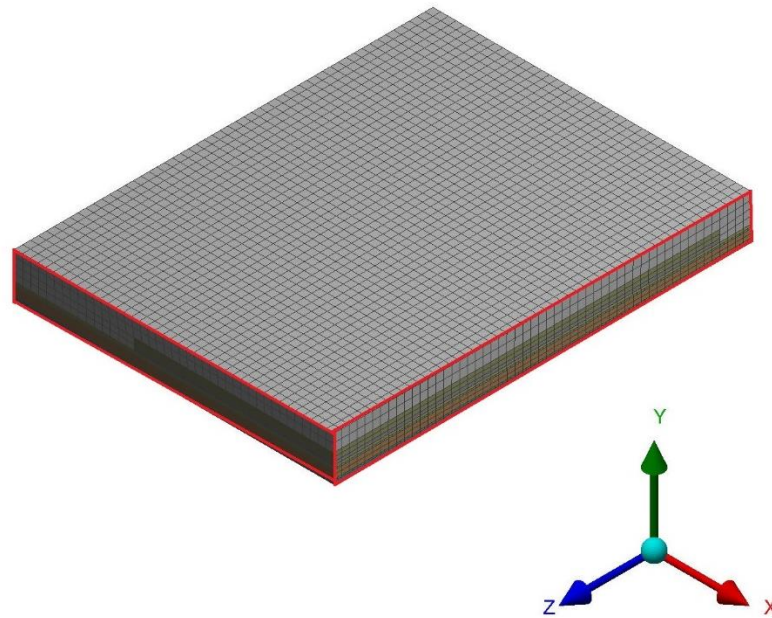


Figure 3.2: Quarter FEA meshed model of LGA

The warpage estimated for the simulated package matches with the reference case study. The maximum out-of-plane deformation was found to be  $39.836 \mu\text{m}$  in the thermal ramp down condition (cooling) (Figure 3.4) and  $53.052 \mu\text{m}$  in the thermal ramp up condition (heating) (Figure 3.5). The – or + sign in the measured value shows the nature of the warpage if it is concave (-) or convex (+).



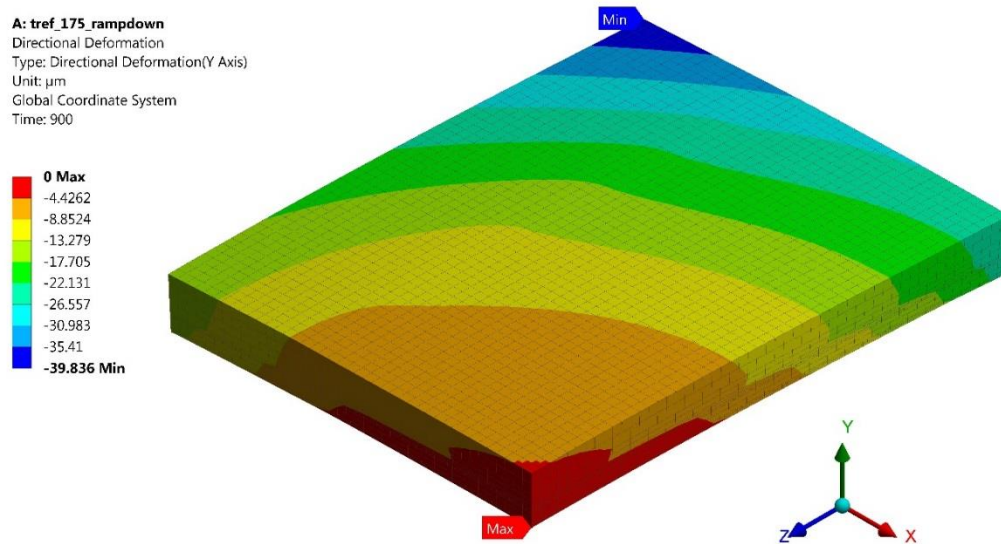


Figure 3.3: Quarter FEA model showing deformation contour measured in  $\mu\text{m}$  on LGA at  $25^\circ\text{C}$

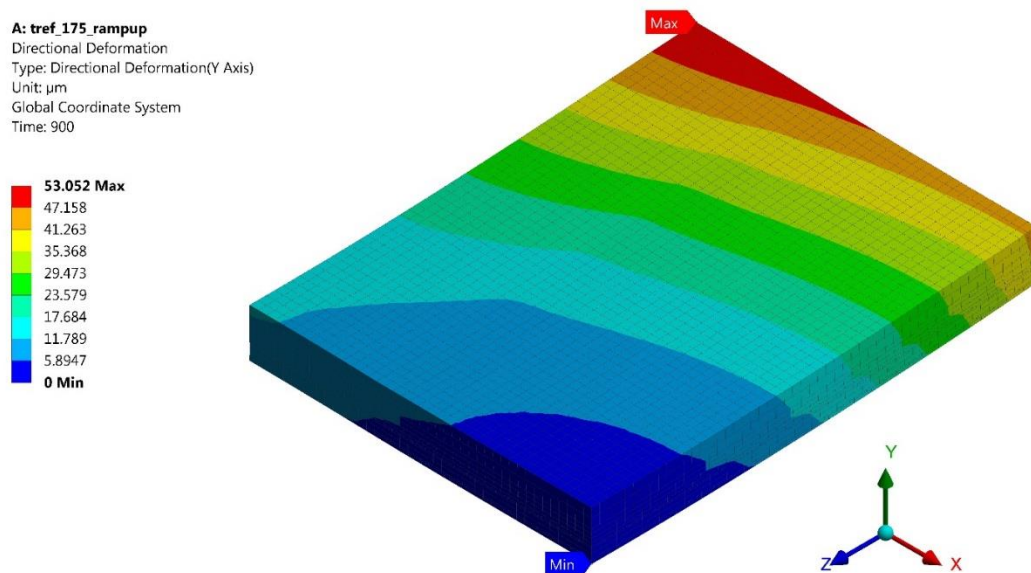


Figure 3.4: Quarter FEA model showing deformation contour measured in  $\mu\text{m}$  on LGA at  $260^\circ\text{C}$

Warpage is measured along the half-diagonal path (from the center to the vertex) at the mid plane of the model as shown in Figure 3.5. Figure 3.6 shows the graph of warpage measured at nodal points across this path. This graph also indicates that the ramp up thermal loading results in a convex type of warpage whereas ramp down thermal loading results in concave type of warpage.

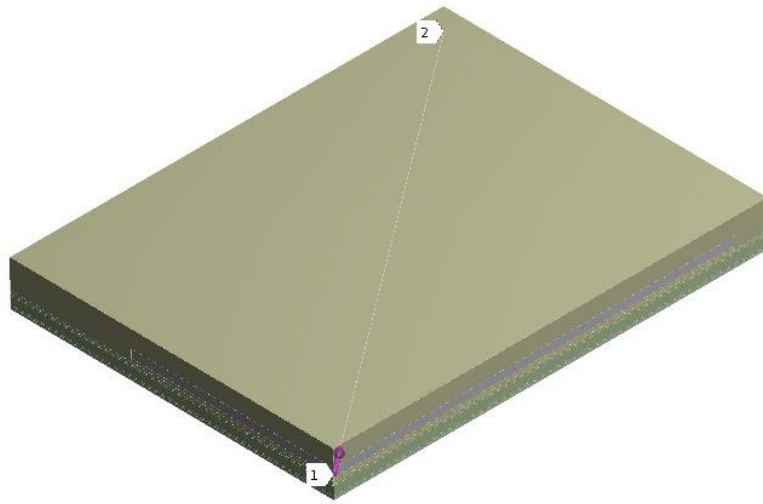


Figure 3.5: Construction Path along center to diagonally opposite vertex in which is useful to measure warpage of the package

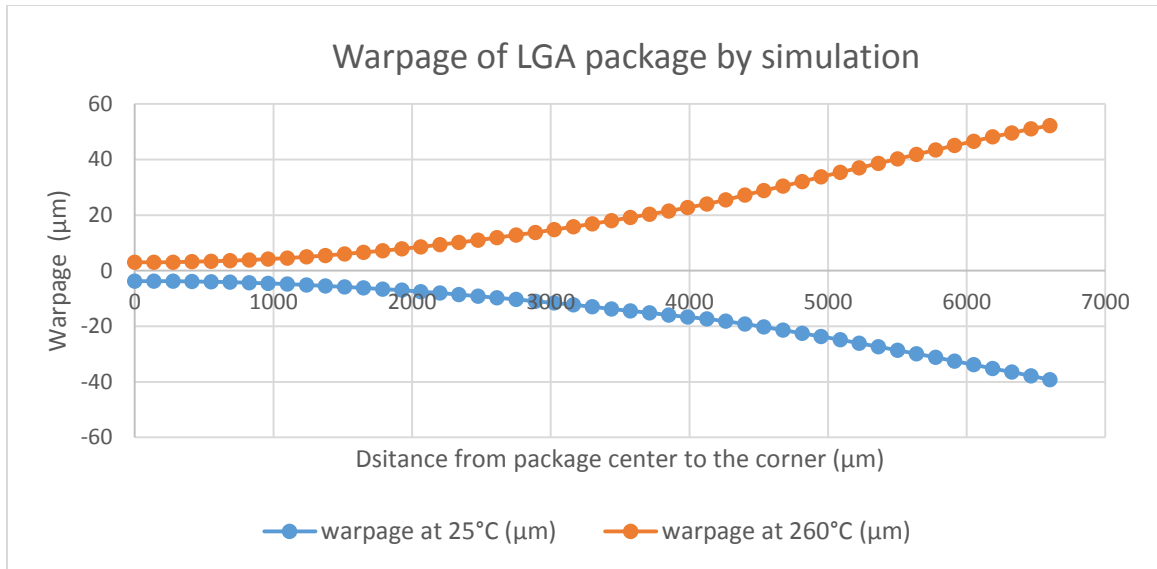


Figure 3.6: Warpage curve along half diagonal of LGA in simulation

### 3.2. Plan of experimentation process for DOE

In this research work, a ball grid array (BGA) flip-chip package is selected having a size of  $10\text{mm} \times 10\text{mm} \times 1.186\text{mm}$  where the die size is  $3.89 \times 4.43 \times 0.30\text{ mm}$ . The sample package with same dimensions shown in Figure 3.8. Figure 3.8(b) shows the top view of the package without lid whereas Figure 3.8(c) shows top view of package with lid. Figure 3.7 represents the schematic of the flip-chip BGA package without lid. Figure 3.8 (a) shows the bottom view of sample flip-chip BGA package having  $12 \times 12$  array of solder joints of diameter  $0.45\text{mm}$  and pitch  $0.8\text{mm}$ . These construction details are considered for package used for simulation. Material properties as well as other dimensional details are as shown in Table 3.2

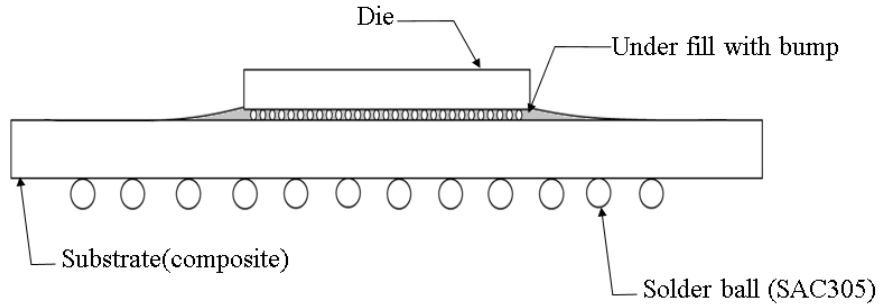


Figure 3.7: Schematic of package of without lid

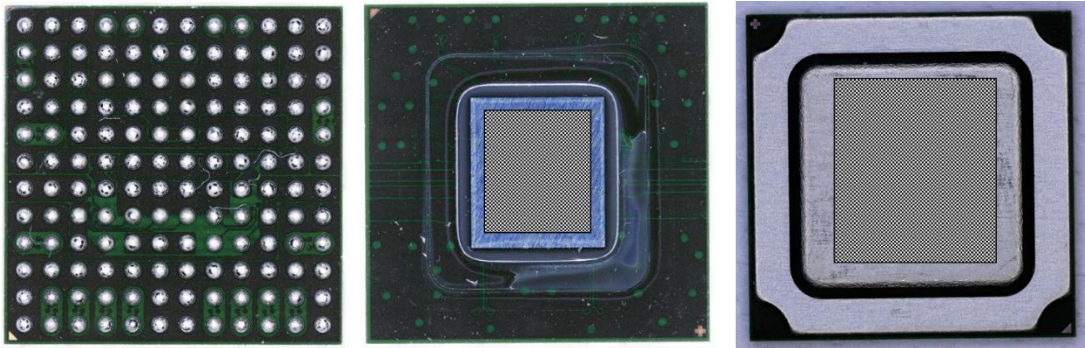


Figure 3.8: (a) Bottom view of the flip chip BGA package (b) Top view of the package without lid (c) Top view of the package with lid

Table 3.2: Dimensional details and Material properties of different layers in package used for experiment

Material layers	Length × Width	Thickness	Young's modulus	Poisson's ratio	CTE	T <sub>g</sub>
Unit	(mm)		(GPa)	-	(ppm/°C)	(°C)
Die	3.89 × 4.43	0.3	131	0.3	2.6	-
PCB	15 × 15	0.5	242	0.11	19.60	
Solder joints	Φ 0.45	pitch 0.8	416	0.35	21.70	
Substrate	10 × 10	0.34	23	0.3	15	
Thermal interface material	3.89 × 4.43	0.12	0.163	0.3	1.92	
Underfill + bump	5.5 × 5.5	0.08	17.41 @ -55 °C, 14.68 @ 225°C	0.35	26.58 (T<T <sub>g</sub> ), 60.84 (T>T <sub>g</sub> )	88

An underfill with micro bumps and a substrate with pre-preg layers are treated as composite materials and their material properties are included in Table 3.2. These are effective material properties calculated by using Equations 2.17, 2.18, and 2.19 as discussed in Section 2.2.2. Figure 3.9 shows the cross section of the lidded package which exposes the composite layers such as substrate with pre-preg and underfill and microbumps under die. This simulation is performed under thermal loading condition as per JESD22-A104D standards, with a ramp rate of 11.33°C/min, one thermal cycle from -40°C to 125°C /hour, and the stress free temperature is 22°C.

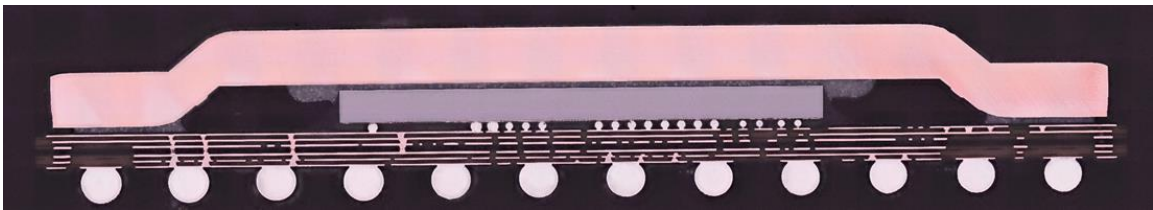


Figure 3.9: Cross section of the lidded package without PCB shows composite layer of underfill- micro bumps and substrate

Based on the package shown in Figure 3.10, a three-dimensional model is generated and used in this experiment. The meshed model is shown in Figure 3.11. Figure 3.12 shows the deformation measured in  $\mu\text{m}$  of the entire package at 125° C that considered for mesh sensitivity analysis. The mesh sensitivity plot in Figure 3.13 shows that as element size decreases, results do stabilize but computational time increases. After the mesh sensitivity analysis was conducted, the element size was fixed as 125 $\mu\text{m}$  for the remaining parametric analyses of the experiment.

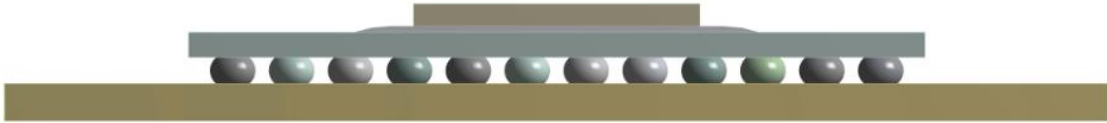


Figure 3.10: Front view of cross section of 3D model of package used in experiment

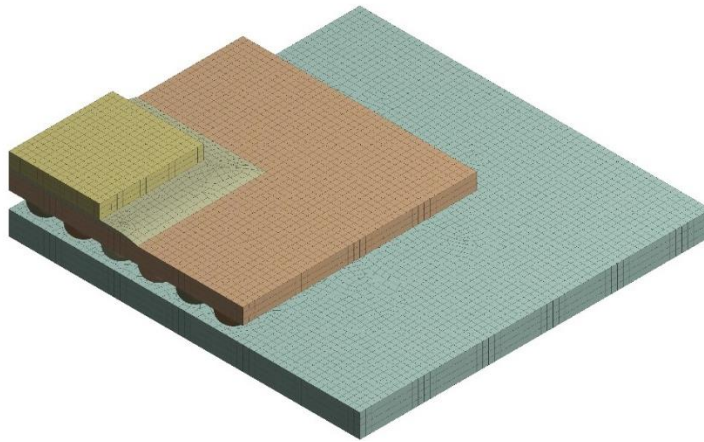


Figure 3.11: Meshed model of the package

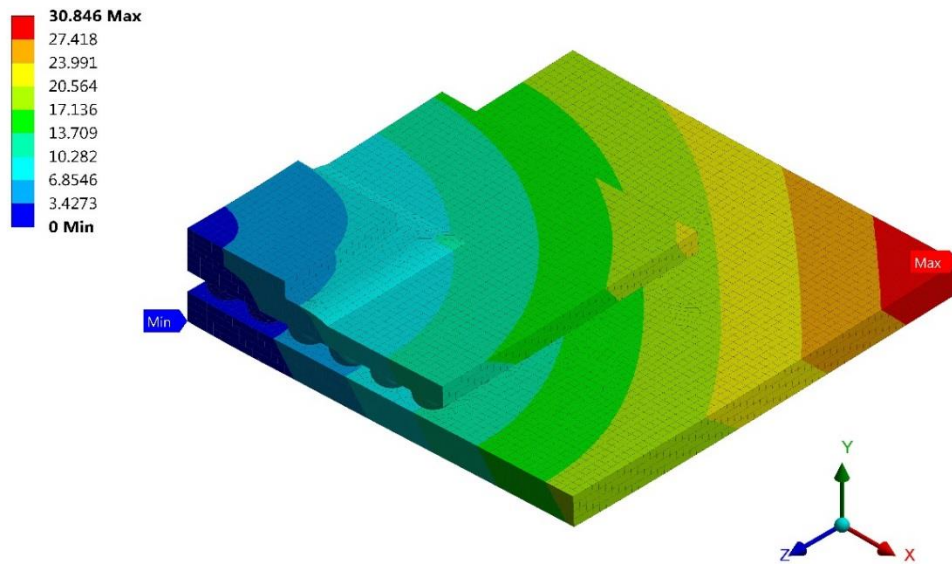


Figure 3.12: Deformation measured in  $\mu\text{m}$  for mesh sensitivity study

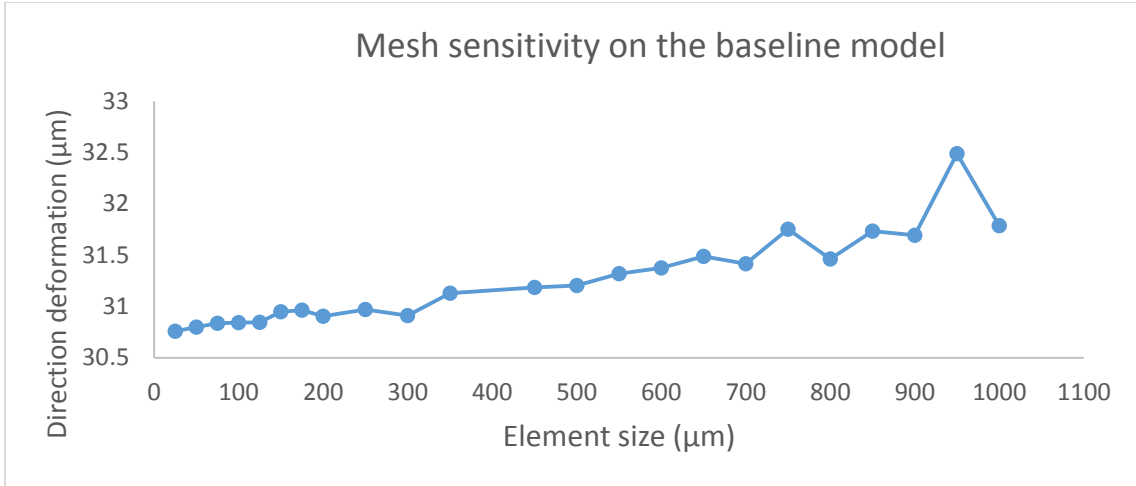


Figure 3.13: Mesh sensitivity on the baseline model

After the meshing parameters and design parameters for the baseline case are finalized, different shape and materials are considered for the design of the lid over package. Different shapes are considered as shown in Figure 3.14.

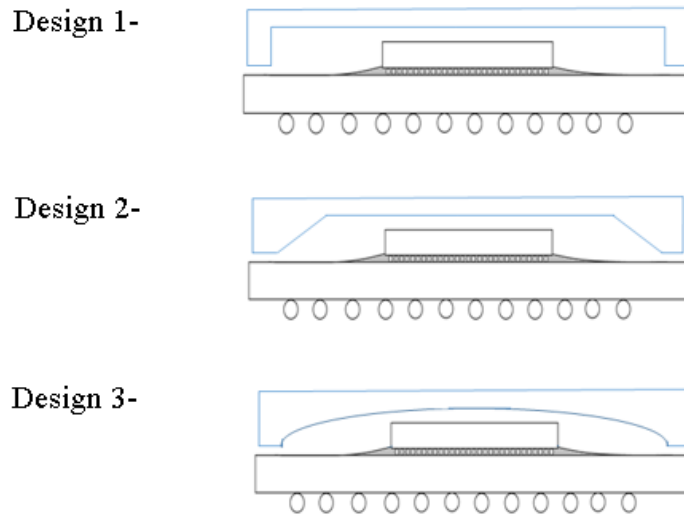


Figure 3.14: Different shapes considered for lid

Materials considered for the lid are as shown in Table 3.3.

Table 3.3: Material properties considered for lid

Material #	Young's modulus	Poisson's ratio	Coefficient of thermal expansion
Unit	(GPa)	-	(ppm/°C)
Material 1	129	0.34	17
Material 2	71	0.33	23
Material 3	45	0.35	26
Material 4	193	0.31	17
Material 5	13 in plane, 11 out plane	0.3	8 in plane, 42 out plane
Material 6	118	0.345	17.7

Table 3.4: DOE matrix based on parameters

	Design 1	Design 2	Design 3
Material 1	Response(1-1)	Response(2-1)	Response(3-1)
Material 2	Response(1-2)	Response(2-2)	Response(3-2)
Material 3	Response(1-3)	Response(2-3)	Response(3-3)
Material 4	Response(1-4)	Response(2-4)	Response(3-4)
Material 5	Response(1-5)	Response(2-5)	Response(3-5)
Material 6	Response(1-6)	Response(2-6)	Response(3-6)

Based on these parameters (shape of the lid and choice of material), the DOE matrix was generated as given in Table 3.4. Results will be discussed in Chapter 4. This DOE study will be helpful to understand the correlation between the parameters. Figure 3.15 provides a flowchart of the research experiment that is carried out in this work.



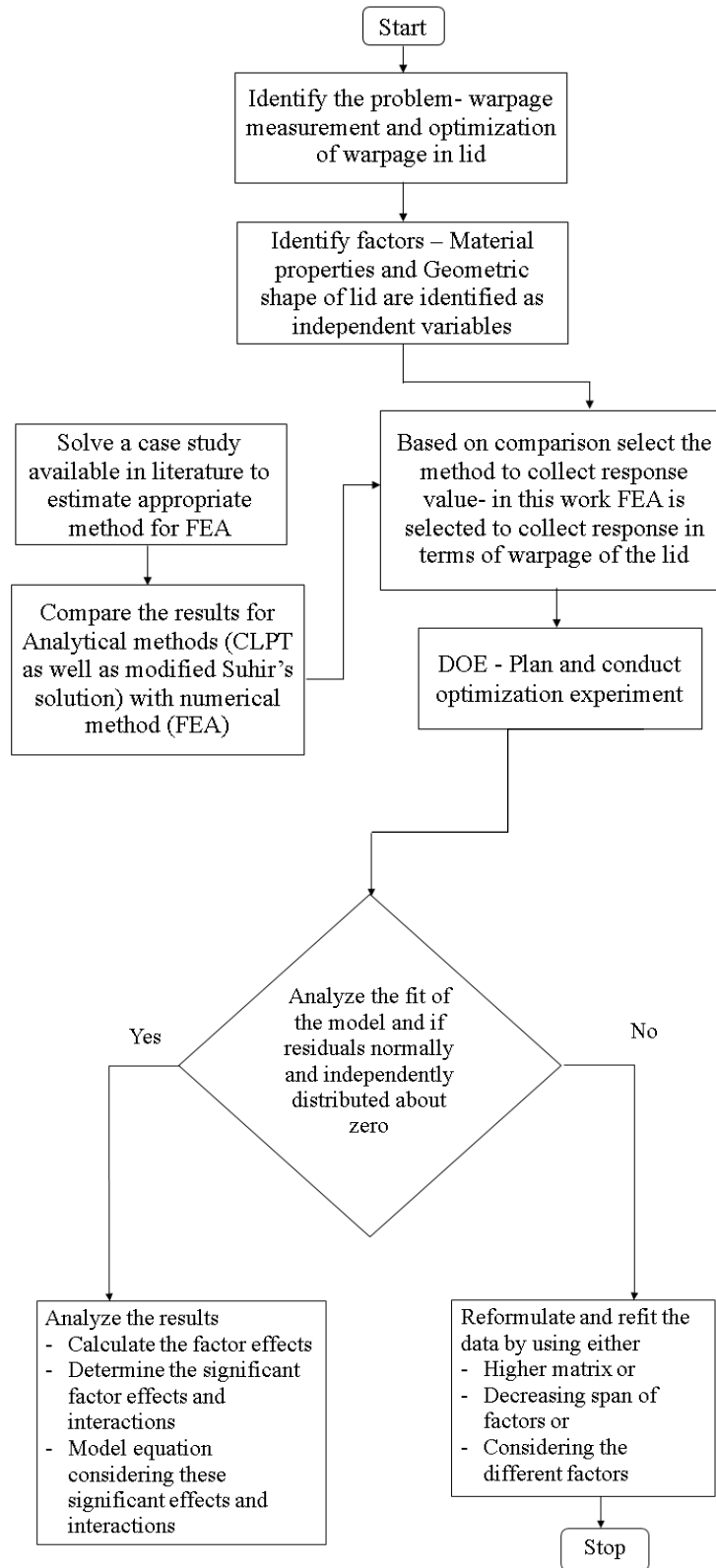


Figure 3.15: Research methodology of the experiment

### **3.3. Summary**

In conclusion, an FEA formulation was developed for generation of warpage data for a subsequent design of experiments analysis presented in Chapter 4. FEA simulations were carried out on 3 different lid designs and 6 different materials leading to a data set of 18 warpage predictions. The material properties that are known to vary across the 6 materials include Young's modulus, Poisson's ratio and coefficient of thermal expansion. Among the, the young's modulus varied the most from 11 GPa in the lowest case to 193 GPa in the highest. The CTE data varied within an order of magnitude between the best and worst cases. Assuming that the loading on the package lid leads to an induced warpage that is within the elastic region, these predictions are reliable. A mesh sensitivity study showed that computational time could be optimized and a mesh size / parameter of '125 $\mu$ m' was selected such that any further reduction in mesh size would lead to no change in the warpage prediction.

## **4. Results and Discussion**

As discussed in the previous chapters based on the LGA case study, experiment of simulation carried out. This experiment involved 6 type of material parameters and 3 types of design parameters. Total 18 simulation runs performed excluding baseline case of package without lid. This baseline is used only for mesh sensitivity analysis to avoid effect of parameters related to lid which are variables on response value of warpage. In this chapter, Comparison of the warpage evaluated by using FEA and analytical methods and design of experiment study is discussed as follows.

### **4.1. Comparison of results for analytical methods and FEA method**

The analytical model of the CLPT three layer of package consists of the die, underfill and microbumps, and the substrate layer. FEA simulation is performed using these three layers and the value of warpage along the X-direction is obtained. Figure 4.1 shows the meshed quarter model and illustrates the three layers. Symmetry and boundary conditions are provided in Chapter 3. Figure 4.2 shows the correlated FEA results of the warpage in the X-direction. Since the package is approximately symmetric, the warpage in the Z-direction is expected to be similar.

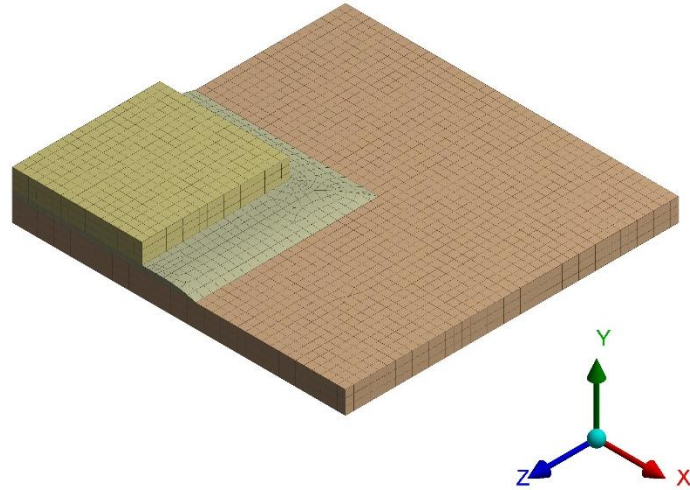


Figure 4.1: Isometric view of meshed quarter model of package considering there layers before solder and reflowing process

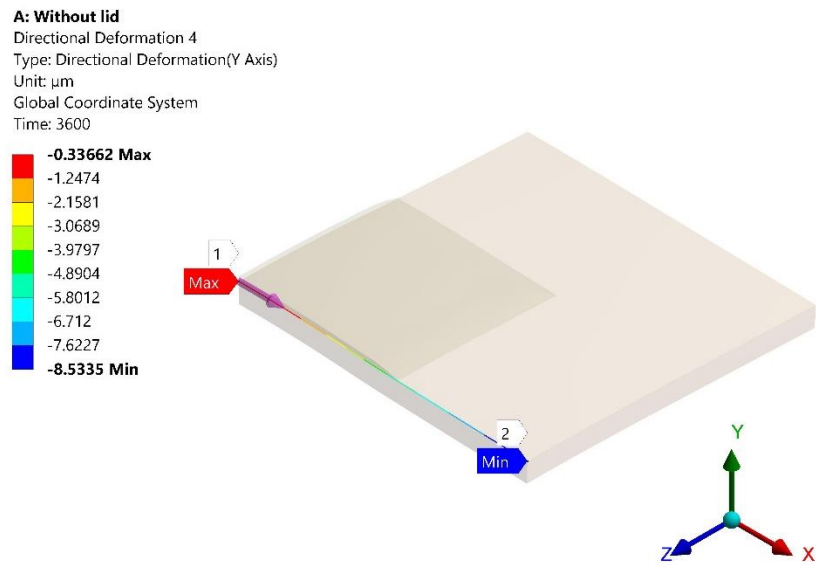


Figure 4.2: Warpage measured in  $\mu\text{m}$  along the mid plane of the package in X-direction

As per the analytical methods discussed in Section 2.2.2, values of the material properties and thickness of the material are utilized in the equations. Warpage is calculated in the X-direction using Equations 2.2 - 2.27.

Figure 4.3 shows that analytical results provide a fall-off shaped curve in which the effect of CTE of underfill material is considered insensitive to stresses and neglected due overall to small thickness of the underfill layer. Whereas the FEA result provides an s-shaped curve which seems more accurate as it considers the effect of the underfill.

In this curve, it is also evident that in classical laminate plate theory that assumes radius of curvature is constant with respect to Krichoff-love hypothesis. . This hypothesis is used to determine stress and deformation of thin layers when those are subjected to force and moment. The assumptions of this hypothesis that are the thickness of the layer is constant during deformation and straight lines normal to mid surface remains normal and straight after the deformation. This leads to more error in calculating response in terms of warpage. Due to this error, the classical laminate plate theory over-predicts the out-of-plane deformation by 40%. This highlights the substantially higher accuracy of FEA method that is presented in this work.

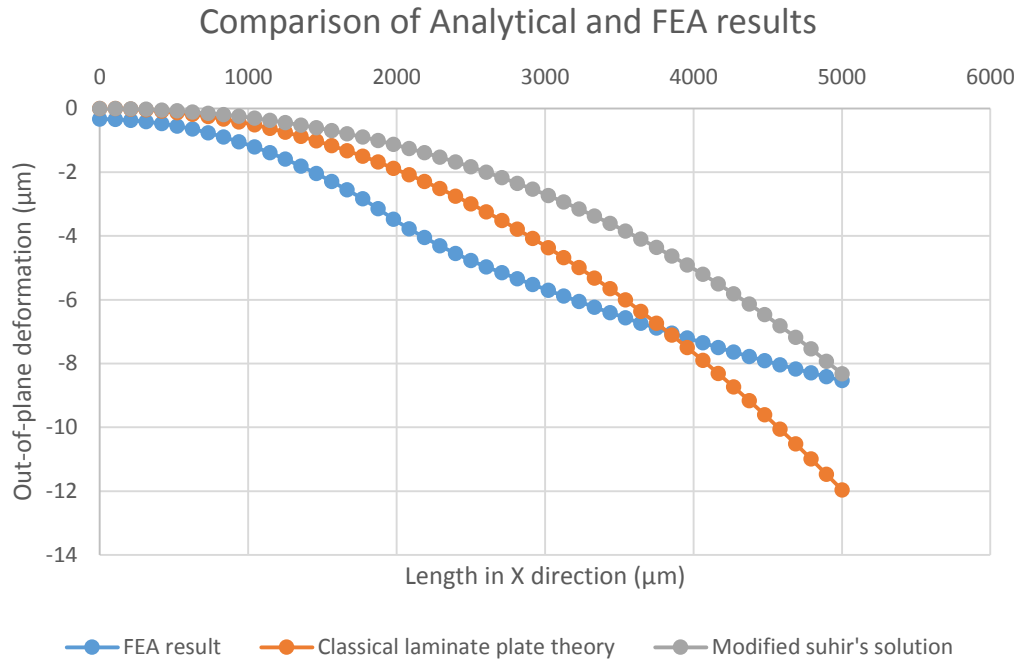


Figure 4.3: Comparison of the Warpage measured by all techniques along the mid plane of the package in the X-direction

#### 4.2. Design of experiment analysis

After the FEA simulation is run for different material sets and different design sets,

Table 4.1 provides the DOE matrix that is now available.

Table 4.1: DOE matrix obtained by parametric study in simulation

Max. Warpage on lid in µm at 125°C			
	Design 1	Design 2	Design 3
Material 1	10.610	10.362	5.357
Material 2	8.964	8.908	4.898
Material 3	9.295	9.437	-7.595
Material 4	10.128	9.881	4.894
Material 5	26.143	26.220	24.691
Material 6	9.851	9.579	4.629

A two-way analysis of variance (ANOVA) is performed to understand if there is any interaction between two independent parameters (material and design) on the

dependent variable (warpage). For this analysis, the null hypothesis for a main effect is that the warpage means for all factor levels are equal and an interaction effect is that the warpage mean for the level of material factor does not depend on the value of the other levels of design factor.

In this analysis, the p value is less than the significance level as shown in Figure 4.5, indicating that the level of material and design factor is associated with warpage of the lid. Figure 4.4 shows different plots obtained using the ANOVA technique. In Figure 4.4 (a), normal probability plot is used to verify the assumption that the residuals are normally distributed since the residual values approximately follows a straight line, excluding few outliers. In Figure 4.4 (b), the residuals versus fits plot provides an indication that data have a non-constant variance. In Figure 4.4 (d), the residuals versus order plot is falling randomly around the center line, but patterns in the near points indicate that residuals near each other may be correlated.

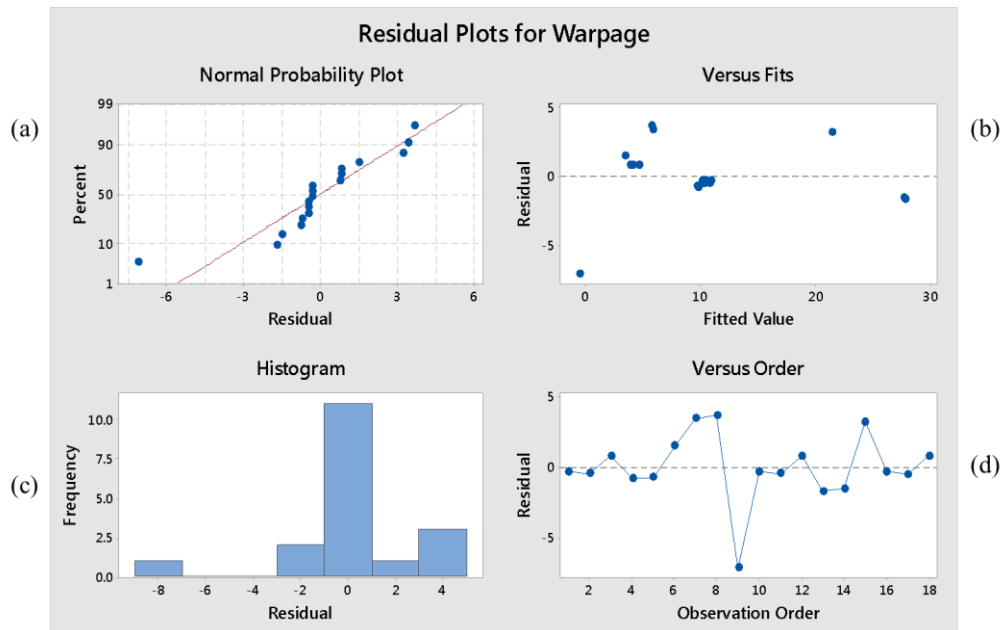


Figure 4.4: Various plots obtained using ANOVA technique

## General Linear Model: Warpage versus Material, Design

### Method

Factor coding (-1, 0, +1)

### Factor Information

Factor	Type	Levels	Values
Material	Fixed	6	1, 2, 3, 4, 5, 6
Design	Fixed	3	1, 2, 3

### Analysis of Variance

Source	DF	Adj SS	Adj MS	F-Value	P-Value
Material	5	896.81	179.362	18.39	0.000
Design	2	158.92	79.462	8.15	0.008
Error	10	97.53	9.753		
Total	17	1153.26			

### Model Summary

S	R-sq	R-sq(adj)	R-sq(pred)
3.12295	91.54%	85.62%	72.60%

### Coefficients

Term	Coef	SE Coef	T-Value	P-Value	VIF
Constant	10.347	0.736	14.06	0.000	
Material					
1	-1.57	1.65	-0.95	0.362	1.67
2	-2.76	1.65	-1.68	0.125	1.67
3	-6.63	1.65	-4.03	0.002	1.67
4	-2.05	1.65	-1.24	0.242	1.67
5	15.34	1.65	9.32	0.000	1.67
Design					
1	2.15	1.04	2.07	0.066	1.33
2	2.05	1.04	1.97	0.077	1.33

### Regression Equation

$$\text{Warpage} = 10.347 - 1.57 \text{Material}_1 - 2.76 \text{Material}_2 - 6.63 \text{Material}_3 - 2.05 \text{Material}_4 + 15.34 \text{Material}_5 - 2.33 \text{Material}_6 + 2.15 \text{Design}_1 + 2.05 \text{Design}_2 - 4.20 \text{Design}_3$$

### Fits and Diagnostics for Unusual Observations

Obs	Warpage	Fit	Resid	Std Resid
9	-7.60	-0.49	-7.11	-3.05 R

R Large residual

Figure 4.5: ANOVA data analysis result



In addition to two-way ANOVA, the DOE is performed using a general full factorial analysis. This analysis considers the interaction between the material and design parameters and results in missing values of p and F. This is due to the use of a 2-level design with one replicate, including all the terms in the model. To remedy this situation, the model is re-fit neglecting the interaction term between the material and design factors. The Pareto chart in Figure 4.6 shows that the interaction between material and design parameter is not significant. If the material parameter is compared with design then it indicates that the material parameter is more significant than the design parameter. The interaction plot between material and design parameter in Figure 4.7 shows near parallelism. This supports the assumption of an absence of interaction between two parameters. The main effects plot with warpage in Figure 4.8 shows that Material 5 has the highest response value where Design 3 has the lowest. This might be interpreted as the Design 3 shape causes the minimum warpage and Material 5 is more elastic in nature and gives maximum warpage. It is difficult to predict the plastic/permanent damage considering only elastic behavior of the material. It can be interpreted from the main effects plot that since the curvature of the lid is smoother, then the deformation is minimum.

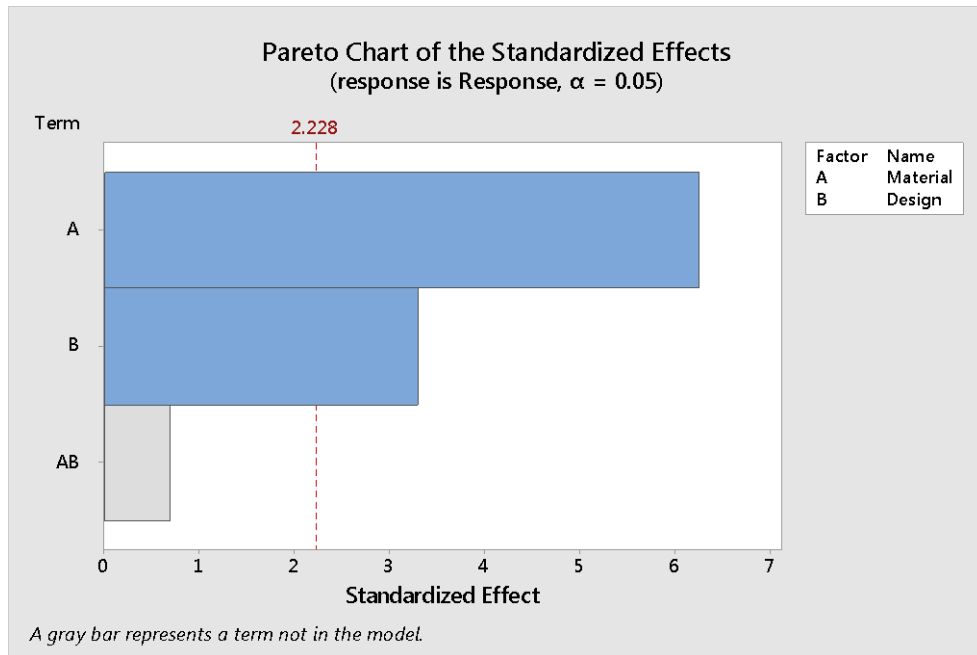


Figure 4.6: Pareto chart highlighting the significance of the material and design parameter

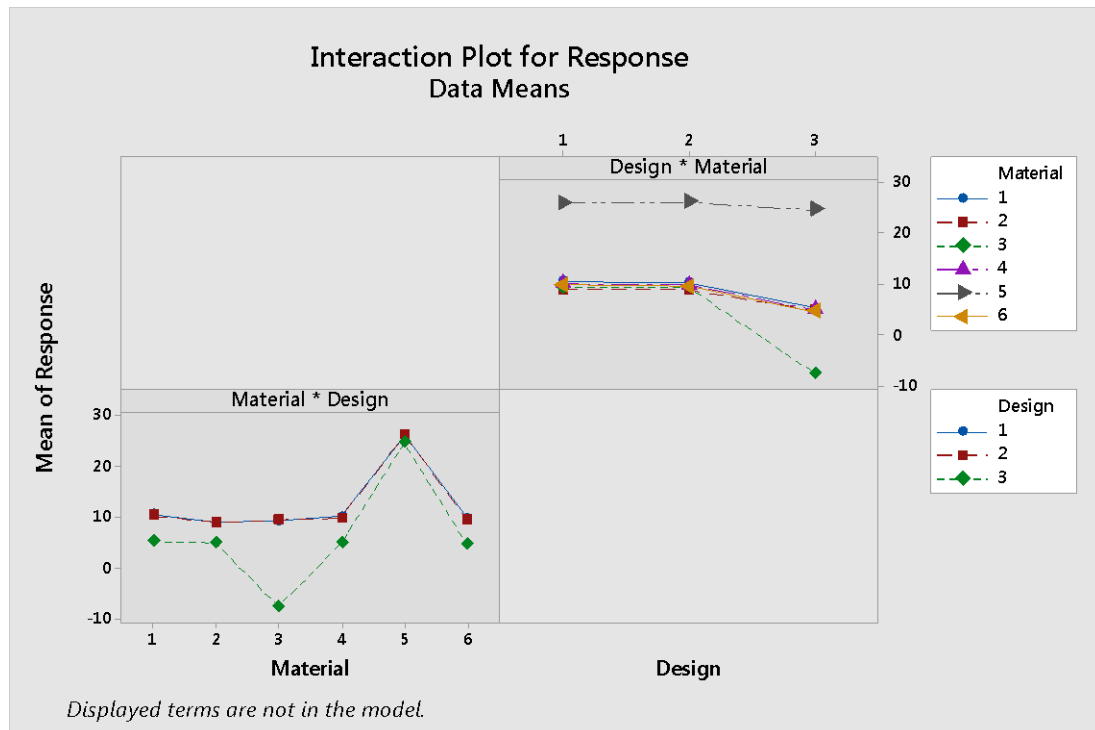


Figure 4.7: Interaction plot considering all the simulation cases for DOE

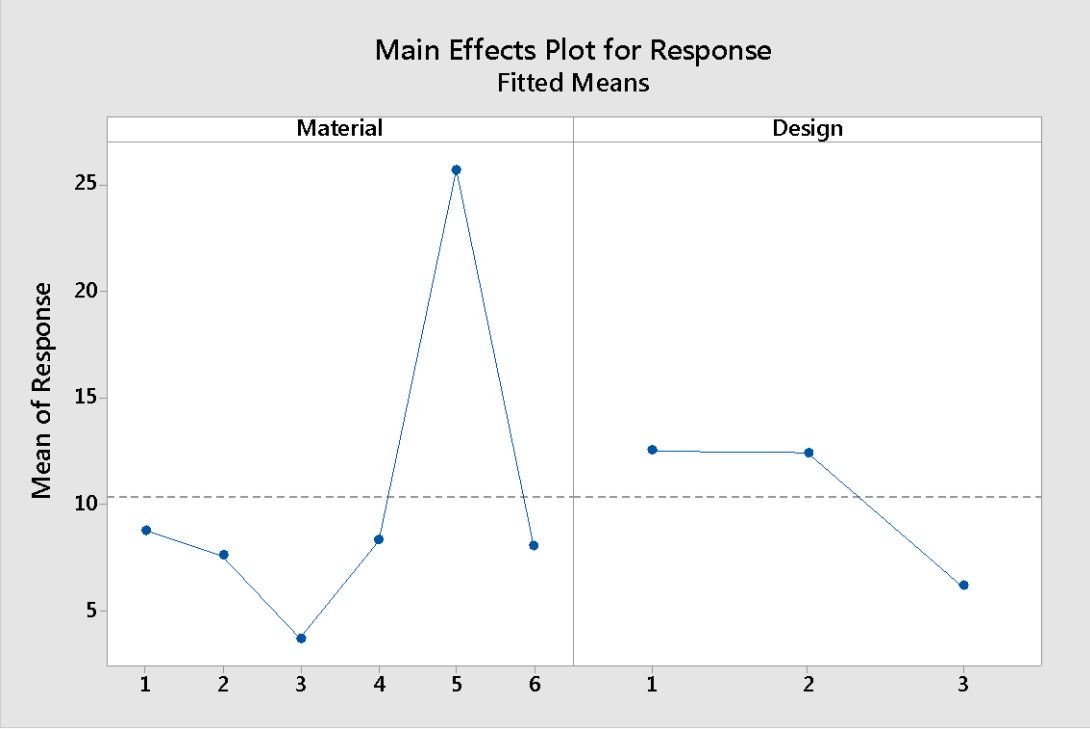


Figure 4.8: Main effects plot of the two parameters individually with respect to mean of response

**4.3. Summary**

After applying both analytical methods successfully on three layer assembly of the package and comparing it with FEA simulation, FEA results seems more accurate. Assumption made based on the literature survey are correct. The design of experiment analysis shows that there is no significant interaction between the two independent variables considered such as material of the lid and design of the lid. If results are analyzed considering only design variable then it can be said that as the curvature of the lid becomes smooth, warpage deformation reduces. It is also observed that type of material plays more significant role in warpage deformation of the lid than the shape of the lid. In this simulation study elastic material properties considered for the package such as Young's

modulus, Poisson's ratio and Coefficient of thermal expansion. Out of these three material properties coefficient of thermal expansion should be more compliant to other layer in the package in order to reduce warpage. Experimental procedures to measure warpage may prove more accurate than analytical and numerical methods. Since ultimate strength and plastic behavior of the material will be tested while thermal loading conditions applied. Although, thermal loading will be accelerated, due to fatigue and creep phenomenon involved warpage result will account for reliability in real life conditions. This is difficult to achieve by FEA tools or analytical tools. So experimental procedure of measuring this parametric study is important in future work to gain more accurate predictions.

## **5. Conclusions and Future Work**

In this thesis, a DOE analysis is conducted on simulated data from warpage of a lidded electronics package using an FEA approach. Warpage in a lid of an optical electronics package can detrimentally affect the reliability of the package as well as its optical performance. The present study focuses on the variety of materials and designs of lids relevant to recent technologies in electronics packaging.

The thesis begins with a review of various methods for warpage measurements demonstrated in the literature, including experimental, numerical, and analytical methods. In a detailed review, various experimental methods are compared based on the properties such as resolution, cost, accuracy, capacity for in-line measurement, flexibility, etc. It is evident that Shadow moiré is the most suitable procedure while digital fringe projection is the second best choice. Other methods, such as digital image correlation, projection moiré, Twyman–Green interferometry, and Fizeau interferometry, have similar performance; but, based on the application, these methods are selected by users. Next, a brief overview of the classical laminate plate theory is presented followed by a literature review of previous work using numerical techniques applied to thermo-mechanical analysis of electronics packages.

In Chapter 3, a FEA is formulated to generate data for this study, owing its tremendous versatility and accuracy in predicting deformations of simple geometries such as those considered herein. The boundary conditions applied to the package geometry exploit its symmetry. The measured value for the warpage on LGA matches with case study recreated as per reference. Additionally, a mesh sensitivity study is also conducted which leads to a significant reduction in computational time and resources.

In Chapter 4, data and results from the FEA are presented and compared to analytical calculations using the CLPT. Finally, these data are used to conduct a DOE analysis to investigate the influence of 3 distinct designs and 6 material choices on warpage of a lid. The key findings from this work can be summarized as follows:

1. Comparisons of warpage estimated from FEA and analytical methods (either by CLPT or modified Suhir's theory) show that FEA results are more accurate as they account for the performance of die attach/underfill materials, regardless of the small thickness of the layer.
2. Between CLPT and modified Suhir's theory, the latter is more accurate since it does not consider the radius of curvature of deformation to be constant.
3. The DOE analysis indicates that there is no significant interaction between the two parameters (lid design and material) considered to affect the warpage in the lid. Hence, they can be considered independent variables in the experiments.
4. Material properties of the lid have a more substantial effect on the warpage of the lid as compared to variability in lid design.
5. It is worth mentioning that these FEA simulations are performed considering material behavior within the elastic limit. In some situations, plastic deformation may occur

which is more permanent and can be measured more accurately via experimental techniques.

Now, a description of some areas of suggested future work is described. Experimental methods can be employed to measure warpage in electronic package lids to generate more robust data for DOE analyses. Such data could better account for plastic behavior of materials as compared to FEA. This is difficult in FEA because plasticity data are not easily available from manufacturers for these materials. Secondly, since the package was approximately symmetric, the behavior in the X-direction was assumed as same for the behavior of other in-plane directions i.e. Z-direction. Verifying this assumption may be of importance. Another approach could be to measure data such as ultimate strength or visco-plastic parameters of these materials and feed them back into FEA analysis in the plastic region.

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